## DIFFRACTION OF AN ELASTIC WAVE IN A DISC

## A. M. Skobeev\*

A numerical solution is constructed for the axisymmetric problem of the diffraction of a plane longitudinal wave in a rigid disc (cylinder) of finite thickness. The disc is enclosed in an unbounded elastic medium; at the contact surface, the tangential stresses are limited by some constant. The incident wave moves along the axis of the cylinder and has the form of a semiinfinite washed-out step. At the same time, a solution is obtained to the corresponding static problem. A study was made of the dependence of the rate of motion of the cylinder and the stress field on the parameters of the problem. In particular, it is shown that the contact conditions have a considerable effect on the stress field only near the lateral surface. The results obtained can be useful for evaluating the errors in measurement of the stresses and velocities in an elastic medium, and possibly also in certain other cases.

1. Within the framework of the dynamic theory of elasticity, this paper considers the axisymmetric problem of the interaction between a longitudinal wave and a rigid cylinder of finite dimensions. The cylindrical coordinates z and r are used; the z axis coincides with the axis of the cylinder, which occupies the region  $-H/2 \le z \le H/2$ ,  $r \le 1$  (Fig. 1). The region outside of the cylinder is filled by an elastic medium. The measurement units were chosen so that the radius of the cylinder, the density of the medium, and the rate of propagation of longitudinal waves in the medium were equal to unity.

In what follows, the following basic designations will be used:  $\rho$  is the density of the cylinder;  $\mu$  is the shear modulus of the medium; t is the time; u(t, r, z) and w(t, r, z) are displacements along r and z; v(t) is the velocity;  $w_0(t)$  is the displacement of the cylinder;  $\sigma_{ZZ}$ ,  $\tau = \sigma_{rZ}$  are the components of the stress tensor, connected with the displacements by Hooke's law

$$\sigma_{zz} = w_z + (1 - 2\mu) (u_r + u/r), \ \tau = \mu (u_z + w_r) (1.1)$$
(1.1)

In addition, the following auxiliary designations are introduced:  $\Gamma$  is the contour of the cylinder in the coordinates r, z;  $Q_1$  and  $Q_2$  are the displacements of the cylinder along the normal and the tangent to  $\Gamma$ ;  $Q_1 = 0$ ,  $Q_2 = w_0$  at the lateral surface;  $Q_1 = w_0$ ,  $Q_2 = 0$  at the bases;  $q_1$  and  $q_2$  are the displacements in the medium at the boundary with the cylinder,  $q_1$  along a normal to  $\Gamma$ ,  $q_2$  along the tangent; a dot denotes a partial derivative with respect to t.

Outside of the cylinder, the dynamic equations of the theory of elasticity are satisfied for u and w

$$u_{ll} = \mu u_{zz} + u_{rr} + (1 - \mu) w_{rz} + u_2 / r - u / r^2$$

$$w_{ll} = w_{zz} + \mu w_{rr} + \mu w_r / r + (1 - \mu) (u_{zr} + u_z / r)$$
(1.2)

The equation of motion of the cylinder is

$$\rho \frac{H}{2} \frac{d^2 w_0}{dt^2} = \int_{\Gamma} r \sigma_{zz} dr + \tau dz$$
(1.3)

Equations (1.2) and (1.3) represent a boundary-value problem with initial conditions at t = 0 and boundary conditions at  $\Gamma$ .

## \*Deceased.

Moscow. Translated from Zhurnal Prikladnoi Mekhaniki i Teknicheskoi Fiziki, No. 3, pp. 139–150, May-June, 1972. Original article submitted February 7, 1972.

© 1974 Consultants Bureau, a division of Plenum Publishing Corporation, 227 West 17th Street, New York, N. Y. 10011. No part of this publication may be reproduced, stored in a retrieval system, or transmitted, in any form or by any means, electronic, mechanical, photocopying, microfilming, recording or otherwise, without written permission of the publisher. A copy of this article is available from the publisher for \$15.00.



The initial conditions describe a plane longitudinal wave, falling from infinity on the upper base of the cylinder. Ahead of the leading front of the wave, the medium is quiescent and not loaded; behind the front, there is a monoaxial deformation with  $\sigma_{ZZ} = 1$ . These conditions have the form

$$w_0(0) = 0, \ v(0) = 0, \ u(0, r, z) = u_t(0, r, z) = 0$$

$$w_z(0, r, z) = w_t(0, r, z) = f((H + T - 2z) / T)$$
(1.4)

Here

$$f(\xi) = 0 \text{ at } \xi \ge 1, f(\zeta) = 1 \text{ at } \xi \le -1, f(\zeta) = (1 + \xi^2 \operatorname{sign} \xi) / 2 - \xi$$
  
at  $-1 < \xi < 1$  (1.5)

The parameter T, entering into (1.4), characterizes the degree of washing-out of the wave.

Fig. 1

At the surface of the cylinder, there are two boundary conditions. The first condition arises from the assumption that the cylinder is rigid. It has the form  $q_1 = Q_1$  at  $\Gamma$ .

The second condition corresponds to a simplified law of dry friction

$$\dot{q}_2 = \dot{Q}_2, \quad \text{ec.ra} \quad |\tau| < k$$
  
 $\tau = k \operatorname{sign} (\dot{q}_2 - \dot{Q}_2), \quad \text{if} \quad |\tau| = k$ 
(1.6)

These conditions differ from Coulomb's law in that the quantity k introduced into (1.6) characterizing the adhesion of the medium to the surface, does not depend on the normal stress.

It must be noted that, if k = 0, from (1.6), it follows that  $\tau = 0$ , which corresponds to slippage conditions. At sufficiently large values of k,  $\dot{q}_2 = \dot{Q}_2$ , which corresponds to adhesion conditions. In these important partial cases, the problem becomes linear.

The problem formulated arises naturally with the study of the effect of shock loading on a body containing a rigid cylindrical inclusion. It is clear that, depending on the situation, different data with respect to the phenomenon may be necessary; it is therefore, expedient to fix the physical interpretation of the problem.

In what follows, it will be assumed that the cylinder is a stress pickup, enclosed in an unbounded elastic medium. The sensing element occupies a certain part of the upper base of the cylinder and does not affect the stress field. It is assumed that the measured stress lies between the maximal and minimal normal stresses acting on the sensing element. The object of the measurement is, obviously to obtain information on the stress in the wave; the main interest lies in the difference between the measured stress and the stress in the incident wave. Therefore, in what follows, the principal emphasis will be on a study of the effect of the parameters of the process on the distribution of the normal stresses at the upper base of the cylinder.

2. The formulated problem was solved numerically. The equations of motion and the boundary conditions were replaced by finite-difference relationships, and the system of equations obtained was solved on a BÉSM-3SM digital computer. It is clear that, under these circumstances, a solution may be obtained only in a finite region. A perturbation, brought about by the motion of the cylinder, for finite values of t, is propagated to a finite region, outside of which the solution has the form

$$u = 0, w = f(t - z + H/2)$$
 (2.1)

It appears natural to calculate the solution in this region; however, the technical characteristics of the machine did not permit using this method. Therefore, additional boundary conditions were introduced with  $z = \pm z_0$  and  $0 \le r \le R$ , r = R and  $-z_0 \le z \le z_0$ , simulating conditions at infinity. It was assumed that, at the boundary introduced, the perturbed motion is close to one-dimensional. The boundary introduced is designated  $\Gamma_1$  (Fig. 1). For the normal and tangential displacements of the perturbed motion toward the new boundary, the previous designations  $q_1$  and  $q_2$  are used; the normal to the boundary is denoted by l. If we define  $C_1 = 1$ ,  $C_2 = \sqrt{\mu}$ , then, at  $\Gamma_1$ , the one-dimensional equations of the theory of elasticity will be satisfied

$$q_{\alpha tt} = C_{\alpha}^2 q_{\alpha ll}, \quad \alpha = 1,2$$

(2.2)

In (2.2), there is no summation with respect to  $\alpha$ ; the subscript *l* denotes the derivative along the normal to  $\Gamma_1$  and  $q_1 = w - f(t - z + H/2)$ ,  $q_2 = u$  at the upper and lower sections;  $q_1 = u$ ,  $q_2 = w - f(t - z + H/2)$  at the lateral section.

The general solution of any of Eqs. (2.2) consists of the sum of two arbitrary functions, one of which describes a wave going away toward infinity, and the other a wave coming from infinity. Since the perturbed motion contains only waves of the first type, the boundary conditions must intersect waves of the second type. Such boundary conditions are the following analog of the Sommerfeld principle of radiation:

$$q_{at} + C_a q_{al} = 0 \quad \text{at} \quad \Gamma_1 \quad (\alpha = 1, 2)$$
(2.3)

At  $\rho = 0$ , Eq. (1.3) degenerates; therefore, using (1.1) and (1.2), we transform it to the form

$$\rho \frac{H}{2} \frac{d^2 w_0}{dt^2} + \iint_{D_1} w_{tt} r \, dr \, dz = \int_{\Gamma_2} r w_z \, dr + \mu r \, w_r \, dz + (1+\mu) \, (1+h/2) \, (u_a - u_b) \tag{2.4}$$

Here, h is a constant;  $\Gamma_2$  is a contour, made up of the straight lines

$$-(H+h)/2 \le z \le (H+h)/2, \quad r = 1 + h/2,$$
  
$$z = +(H+h)/2, \quad 0 \le r \le 1 + h/2$$

where  $D_1$  is the region included between  $\Gamma$  and  $\Gamma_2$ ;  $u_a$  and  $u_b$  are the values of u at the upper and lower corners of  $\Gamma_2$ .

In a cylindrical system of coordinates, there arise conditions at r = 0 which, from considerations of symmetry, are taken in the form

$$u = 0, w_r = 0$$

The initial conditions were set not at t = 0, but at  $t = H/2 - z_0 < 0$ . This comes down to a situation in which the sought functions and their derivatives are equal to zero at  $t = H/2 - z_0$  and  $z \le z_0$ .

Thus, the system of equations (1.2) is solved with the boundary conditions (1.6), (2.3), (2.4) and null initial conditions.

As is usual, the region included between  $\Gamma$  and  $\Gamma_1$  is divided by straight lines parallel to the coordinate axes into squares with the side h. All the functions are calculated only at the nodes of the grid obtained, and for discrete values of the time. It is assumed that  $\Gamma$  and  $\Gamma_2$  pass through nodes of the grid. This means that 1/h, H/h,  $z_0/h$ , and R/h are all whole numbers.

The solution is calculated for successive values of t, with the spacing  $t_0$ , starting from  $t = H/2-z_0 + t_0$ .

In what follows, the arguments of the functions are not calculated; the solution for the moment of time t is called the middle solution, for  $t-t_0$ , it is called the lower solution, and for  $t + t_0$ , it is called the solution for the upper layer. To obtain an approximation of the derivatives at the external points of the region in the middle layer, the central difference operators  $\delta_z$ ,  $\delta_{zz}$ , and  $\delta_{zr}$  are introduced using the formulas

$$\delta_z f_{ij} = (f_{i,j+1} - f_{i,j-1}) / 2h \ \delta_{zz} f_{ij} = (f_{j,j+1} - 2f_{i,j} + f_{i,j-1} / h^2 \ \delta_{zr} f_{ij} = (f_{i+1,j+1} + f_{i-1,j-1} - f_{i,j+1} - f_{i-1,j} / 4h^2 \ (2.5)$$

Here,  $f(\mathbf{r}, \mathbf{z})$  is an arbitrary function of  $\mathbf{r}$  and  $\mathbf{z}$ . Analogously, the operators  $\delta_{\mathbf{r}}$ ,  $\delta_{\mathbf{rr}}$ ,  $\delta_{l}$ ,  $\delta_{ll}$ ,  $\delta_{t}$ , and  $\delta_{tt}$  are introduced, except that in the expressions for the two last operators,  $t_0$  is used instead of  $\mathbf{h}$ . Thus, for the corresponding derivatives, the operators introduced yield an approximation of the second order.

Since Eqs. (1.2) are determined only at external points of the region, for them we can immediately write the difference analogies

$$\delta_{tt}u = \mu \delta_{zz}u + \delta_{rr}u + (1 - \mu) \delta_{rz}w + \delta_{r}u / (hi) - u / (hi)^{2}$$

$$\delta_{tt}w = \delta_{zz}w + \mu \delta_{rr}w + \mu \delta_{r}w / (hi) + (1 - \mu) \delta_{rz}u + (1 - \mu) \delta_{z}u / (hi)$$
(2.6)

The expressions obtained describe a three-layer explicit scheme of the second order of accuracy for the system of equations (1.2). For the scheme to be stable, the Courant criterion must be satisfied [1]; therefore, in what follows, it is assumed that  $t_0 = h/2$ .

The initial conditions for the system of equations (2.6) reduce to the fact that all the functions are equal to zero at  $t = H/2 - z_0 - t_0$  and  $t = H/2 - z_0$ .







Fig. 3

The boundary conditions are of a more complex character. For the conditions at the external boundary, we can use the central difference relationships introduced into (2.5). The difference analogy of conditions (2.3) has the form (no summation with respect  $\alpha$ ; l is the normal to  $\Gamma_1$ )

$$\delta_l q_{\alpha} + C_{\alpha} \delta_l q_{\alpha} = 0 \quad \text{i.a.} \quad \Gamma_1 \quad (\alpha = 1, 2) \tag{2.7}$$

Since central difference relationships are used in (2.7), a point lying outside the region enters into (2.7). To exclude this point, we make use of the fact that, at the boundary,  $q_{\alpha}$  also satisfies Eqs. (2.2). The difference approximation of these equations  $\delta_{tt}q_{\alpha} = C_{\alpha}^{2} \delta_{ll}q_{\alpha}$  contains the same point, lying outside of the region under consideration, as in (2.7). This permits excluding this point and obtaining an approximation of the boundary conditions in the form ( $q_{\alpha}$  is taken in the middle layer)

$$2C_{\alpha}\left(\delta_{l}q_{\alpha}+C_{\alpha}\delta_{l}q_{\alpha}\right)/h+\delta_{ll}q_{\alpha}-C_{\alpha}^{2}\delta_{ll}q_{\alpha}=0$$
(2.8)

Boundary condition (2.4) was also approximated by central difference relationships. The integral entering into the left-hand part is of the order h and can be transformed to the form

$$\iint_{D_1} w_{tt} r \, dr \, dz = \frac{h}{2} \frac{d^2 w_0}{dt^2} + \frac{h}{2} \int_{\Gamma_4} w_{tt} \, dz + O(h^2) \tag{2.9}$$

Here,  $\Gamma_3$  is the segment r = 1, -H/2 < z < H/2.

The integrals with respect to  $\Gamma_2$  and  $\Gamma_3$  were calculated using the trapezoid rule, while the functions under the integral signs were taken at the points of intersection between  $\Gamma_2$  and  $\Gamma_3$ , and the straight lines of the grid. The symbol  $\Sigma$  is used for the corresponding integral sums. Since  $\Gamma_2$  does not pass through nodes of the grid, for an approximation of the first derivatives, the operators  $\delta_Z^{1/2}$  and  $\delta_T^{1/2}$ , analogous to those introduced into (2.5) were used, but with a spacing of h/2.

2

The difference analog of (2.4) has the form

$$\frac{\partial (H+h)}{2} \delta_{tt} w_0 = \sum_{\Gamma_2} r \left( \delta_z^{t/2} w + \mu \delta_r^{t/2} w \right) + \frac{h}{2} \sum_{\Gamma_3} \delta_{tt} w + (1-\mu) \left( 1 + \frac{h}{2} \right) \frac{u \left( t, 1+h, h+H/2 \right) - u \left( t, 1+h, -h-H/2 \right)}{2} + O\left( h^2 \right)$$
(2.10)

Here,  $\delta_{tt}w$  was calculated in the lower layer. In the derivation of (2.10) use was made of the fact that  $u_a$  and  $u_b$  and the left-hand part of (2.9), have an order of magnitude h; thus, for them, an approximation of the first order is suitable.

For the remaining boundary conditions, there are required unilateral operators of the second order, determined in the second layer

$$\delta_z^{1} f = (4f(z+h) - f(z+2h) - 3f(z) / 2h$$
(2.11)

and the analogous operators  $\delta_r^1$ ,  $\delta_l^1$ , and  $\delta_t^1$ ; only in the last operator is  $t_0$  used instead of h.

At r = 0, the condition has the form

$$u=0,\,\delta_{r}^{1}w=0$$

The condition at the boundary of the cylinder  $q_1 = Q_1$  remains unchanged, while condition (1.6) assumes the form (*l* is the normal to  $\Gamma$ ;  $k_1 = k/\mu$ )

$$\delta_{l}^{1}(q_{2} - Q_{2}) = 0, \quad \text{if} \quad |\delta_{l}^{1}q_{2}| < k_{1}$$

$$\delta_{l}^{1}q_{2} = k_{1} \operatorname{sign} \delta_{l}^{1}(q_{2} - Q_{2}), \quad \text{if} \quad |\delta_{l}^{1}q_{2}| = k_{1}$$
(2.12)

The conditions obtained constitute a system of nonlinear equations for determining  $q_2$  at  $\Gamma$ . Each of the equations contains only one unknown; therefore, this system is easily solved. The solution is unique and has the form



Fig. 4



Fig. 5







Here

 $k_2 = 2\hbar k_1 \ / \ \mu, \ \ y_1 = 2\hbar \delta_t^1 \ (q_2 - Q_2) \ / \ 3 - q_2, \ \ y_2 = 2\hbar \delta_l^1 q_2 \ / \ 3 + q_2$ 

 $y_1$  and  $y_2$  do not contain  $q_2$  at  $\Gamma$ .

Thus, the solution in the upper layer is completely determined. Moving from layer to layer, we can construct a solution for any given values of t.

3. The above scheme was put into practice in the form of a program with only slight changes. The calculation was carried out until steady-state conditions had been attained and took from 20 to 60 min; H was varied from 0.5 to 2.0,  $\mu$  from 0.1 to 0.5, k from 0 to 1.0,  $\rho$  from 0.5 to 4.0, T from 0.5 to 1.0,  $z_0$  from 2.5 to 5.0, R from 3.0 to 6.0, h from 0.1 to 0.2. Thus, only the case of large values of H remained uninvestigated. This gap is explained by the fact that, with a fixed spacing with respect to h and r, for large values of H, too many points of the grid are required.

As a result of the calculations it became clear that the most stable characteristic of the process is the velocity of the cylinder v(t), which is practically independent of  $\mu$  and k, and depends only weakly on  $\rho$  and H. The solid line on Fig. 2 shows the dependence of v on t for  $\mu = 0.1$ , k = 0, H = 2.0,  $\rho = 1.0$ , T = 0.25, h = 0.2,  $z_0 = 2.5$ , R = 8.0.

The dotted line corresponds to  $\mu = 0.5$ , k = 1.0, the dash-dot line to  $\mu = 0.5$ , k = 0, while the remaining parameters are the same. Calculations were carried out for intermediate cases ( $\mu = 0.2, 0.3, 0.4$ , and k = 0); the difference obtained was still less. Analogous results were also obtained for H = 0.5, 0.8, 1.0, and 1.6. Since in the theory of elasticity  $\mu \le 0.5$  [2], and at k = 1.0, no slippage was observed, it may be assumed that for H  $\le 2.0$  and  $\rho = 1.0$ , the velocity of motion of the cylinder is practically independent of  $\mu$  and k.

Figures 3 and 4 show the dependence of v(t) on H and  $\rho$ . On Fig. 3, curves 1 and 2 correspond to H = 0.5, 1.0 and  $\rho = 0.3$ ,  $\rho = 1.0$ , k = 0, T = 0.5; on Fig. 4, curves 1, 2, 3, 4 correspond to  $\rho = 0.5$ , 1.0, 2.0, 4.0 and H = 1.0,  $\mu = 0.3$ , k = 0, T = 0.5.

It can be seen that the dependence of the velocity on H and  $\rho$  is brought out clearly and that it corresponds to intuitive concepts.

The dependence of the stress field on the parameters of the problem comes out even more clearly. The essential point in the problem is the value of  $\sigma_{ZZ}$  at the surface of the cylinder. Figure 5 illustrates the value of  $\sigma_{ZZ}$  (t<sub>1</sub>, r, H/2); t<sub>1</sub> is chosen so that steady-state conditions have set in. Curve 1 on this figure shows  $\sigma_{ZZ}$  as a function of r for H = 0.5; curve 2 for H = 1.0; curve 3 for H = 2.0; the other parameters are

$$\mu = 0.3, \ \rho = 1.0, \ k = 0$$

On Fig. 6, curve 1 shows  $\sigma_{ZZ}$  for  $\mu = 0.1$ ; curve 2 for  $\mu = 0.5$ ; the other parameters are

$$H = 2.0, \rho = 1.0, k = 0$$

As an important special circumstance, it must be noted that  $\sigma_{ZZ}$  at the base of the cylinder is practically independent of k, while, at the same time,  $\sigma_{ZZ}$  depends strongly on k at the lateral surface. The dependence of  $\sigma_{ZZ}$  at the center of the upper base of the cylinder on k for  $\mu = 0.1, 0.2, 0.3, 0.4, 0.5$  and  $H = 2.0, t = \infty$ , is shown on Table 1.

Figure 7 shows the dependence of  $\sigma_{ZZ}$  on z at the lateral surface of the cylinder for k = 0.1, 0.3 and H = 2.0,  $\mu$  = 0.3.

The dependence of  $\sigma_{ZZ}$  at the center of the upper base of the cylinder on  $\mu$  and H for k = 0.0,  $\rho$  = 1.0, t =  $\infty$ , is shown on Table 2.



Figure 8 gives the dependence  $\sigma_{ZZ}(t)$  at the center of the upper base of the cylinder for  $\mu = 0.1, 0.5$ and  $H = 2.0, \rho = 1.0, T = 0.25$ .

Figures 9-12 give the distributions of the stress  $\sigma_{ZZ}(\mathbf{r}, z, t)$  for moments of time t = 1.0, 1.5, 2.0, 5.0 and the following combination of parameters

$$\mu = 0.3, \ k = 1.0, \ \rho = 1.0, H = 1.0, \ T = 0.5$$

These figures were obtained with the aid of a special program in which the read-out of an ATsPU-128 computer was used. The coordinate z is plotted along the axis of ordinates, and r along the axis of abscissas. The contour of the cylinder is indicated by the dashed line; the figures show only the region  $|z| \leq$  $(H/2 + z_0)/2$ ,  $0.5 \leq r \leq 1.5$ , although the whole region  $|z| \leq z_0$ ,  $0 \leq r \leq R$  was developed in the computer. The numbers  $1, 2, 3, \ldots$  denote zones of low values, and the letters high values of the stresses, compared to the stresses in the incident wave. The white fields, with the exception of the white field in front of the symbol 9 ahead of the front of the incident wave in Fig. 9, correspond to the stress in the incident wave. A transition to the following symbol corresponds to a change in  $\sigma_{ZZ}$  by 0.05. In particular, the absence of a footnote denotes  $\sigma_{ZZ} = 1.0 \pm 0.025$ , the symbol 1 denotes  $\sigma_{ZZ} = 0.95 \pm 0.025$ , and the symbol A denotes  $\sigma_{ZZ} =$  $1.05 \pm 0.025$ . In Fig. 9, the leading boundary of the wave front is located at the level of the lower base of the cylinder. In the central part of the upper base, the stress is  $\sigma_{ZZ} = 1.10 \pm 0.025$ , which, at r > 0.5, rises with approach to an angular point. Below the lower base, in the central part of the stress is  $\sigma_{ZZ} = 0.90 \pm$ 0.025. In the vicinity of an angular point, zones of stress concentration appear clearly (the symbol A). Along the lateral surface of the cylinder, the unstable character of the stress distribution is clearly marked. On the whole, the stresses here rise from the edges toward the lateral surface, where  $\sigma_{ZZ} = 0.95 \pm 0.025$ .

On Fig. 10, the stress distribution along the lateral surface becomes more symmetrical; however, the character of the distribution of the stresses, i.e., their increase from the edge toward the center, is retained. In the vicinity of angular points, above the upper and below the lower bases of the cylinder, there is an appreciable growth of the stress concentration. The stresses in the central part of the lower base rise ( $\sigma_{z,z} = 0.95 \pm 0.025$ ).

In Fig. 11, the stress distribution immediately along the lateral surface is close to steady-state conditions. Above the upper and below the lower bases, there is an increase in the zones of stress concentration.

Figure 12 corresponds to fully established quasi-steady-state conditions. The concentration of the stresses along the upper and lower bases is identical, and, at  $0 \le r \le 0.5-0.6$ , is equal to  $\sigma_{ZZ} = 1.10 \pm 0.025$ .

		1111111111	1111111111111		1	1111665554453551111	
	Charles and the second	***********	111111111111		11. 444	1116666554433355111	
		11111111111111	1111111111111		111 1	111000022264492255511	
	1111	1111111111111	111111111111	111555555511111	1114	1112222224722255111	
	11111		11111111111111	55555555555555511	11111	111000000000000000000000000000000000000	
	1111111	2211111111111	535111555555	3353355555555555	2111113	11999922224+22255111	
		333511111111	55555555555	555555555555555	1 1111155	1000022224+2225551111	
9	[[[[[[[[[[	55555551111	555555555555	5522222222555555	11111111111115555		
<b>A</b> A		55555555555	55555555555	2222222222222322255	11111111111555555	000022234444223555555111	
AAA		555555555555	555555555555	22222222222222223	111111113355532	0000222200002223333355511	
8444 	22111111111	55555555555555	2255555555555	22222322222222222	5355555555552222	00222200001222555551	
<b>AAAAA</b> A	11111122222	2222222222	21225222222222	*********??????????????????????????????	55555322232224	22224440422223333355555	
<b>AAAAAAA</b> A	1222222111	332223333333333	22222222222222	***********	22222222222222222	222494422222255555555	
	1333555111	111111111111111	22222222222222	************	33444444444444	*****1722233355555393	
JAAAAAAAAA	115555556	33333444444	22222222222222	222222444444422	5555555555555555	+++2222353555532222	
88888888888888	AA 15224422	222244422080	2222222222222	222222224444432	59999999921881198	+22225511111 152242	
99999999999999	117655321 68	222444207888	22222222222222	272222224444442	555559286 688	725555111 948 17421	
2888888887777	78776421ABF	2244426789	22222222222222	222222222444442	55555986	525511 448889412222	
988888777711111111111111111111111111111	987531ATE	825999995	222222222222222	**************	6555526	555 57 JA8894 1122	
1006666777701	003333#////	11111111111	111111111111	1111111111111111	11111111111111111	2211 AA550011022	
1666888877777	166000331				,	TT.110699988 11117	
1009888881111	12722221				,	11112 AAA66666666666	
188888888888	וררררר				,	8999977777 11117	
1999999999999	VLLLLL88				,	999####### IIIII17	
12292222222	1188888B				,	***************************************	
19999999999999					. ,	WWW 11111112	1
1999999999999	. 19989999				,	1111111111	
9999999999999	62888884/				,	111111111122	
19999999999999	1999999999/				,	11111111111122	
19999999999999	12229999991	S. S			,	111111111111111222	
19999999999999	122688888				,	111111111111111112222	
1999999999999	185555555					1111111111111111111111	
19999999999999	122222222				,	525551111111111111111	
19999999999999	19999999/	and the second second			,	535355511111111111111	
19999999999999	188888888				. /	52525255111111111111111	
1888888888888						525555555551111111111111	
1888888888888	*******				,	55555555555511111111111	
122222222222	*******					5355555555555551111111	
							4



Fig. 10

Fig. 9



388

It is of interest to note that, during the period of not-fully established motion (Figs. 9, 10, 11), the effects of the stress concentration in the middle part of the upper and lower bases of the cylinder is less ( $\sigma_{ZZ} = 1.05 \pm 0.025$ ) than after the establishment of a quasi-steady state (Fig.12). This difference, however, is not great, and lies within the limits  $0.05 \pm 0.025$ .

## LITERATURE CITED

- 1. V. Vazov and J. Forsythe, Difference Methods for Solving Differential Equations in Partial Derivatives [Russian translation], Moscow, Izd. Inostr. Lit. (1963).
- 2. I. N. Sneddon and D. S. Berri, Classical Theory of Elasticity [Russian translation], Izd. Fizmatgiz, Moscow (1961).